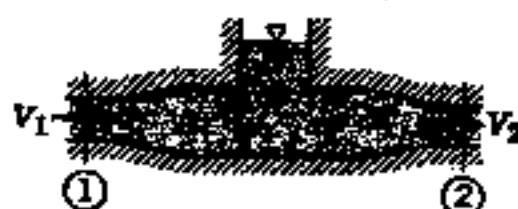


prob.1 A section of pipe carrying water contains an expansion chamber with a free surface whose area is 2 m^2 . The inlet and outlet pipes are both 1 m^2 in area. At a given instant, the velocity at section ① is 3 m/s into the chamber. Water flows out at section ② at $4 \text{ m}^3/\text{s}$. Both flows are uniform. Find the rate of change of free surface level at the given instant. Indicate whether the level rises or falls.

FIGURE I.

hint: Governing eq. $\frac{d}{dt} \int_{CV} \rho dV + \int_{CS} \rho \vec{V} \cdot d\hat{A} = 0$.

prob.2 The sketch shows a vane with a turning angle of 60° . The vane moves at constant speed, $U = 10 \text{ m/s}$, and receives a jet of water that leaves a stationary nozzle with speed $V = 30 \text{ m/s}$. The nozzle has an exit area of 0.003 m^2 . Determine the force that must be applied to maintain the vane speed constant.

FIGURE II.

hint: Governing eq.,

$$\vec{F} = \vec{F}_S + \vec{F}_B = \frac{\partial}{\partial t} \int_{CV} \vec{V}_{xyz} \rho dV + \int_{CS} \vec{V}_{xyz} \rho \vec{V}_{xyz} \cdot d\hat{A}$$

$$\rho_{H_2O} = 999 \text{ kg/m}^3.$$

Prob. 3
25 points

Water flows from the pipe shown in Fig. III. as a free jet and strikes a circular flat plate. The flow geometry shown is axisymmetrical. Determine the flowrate and the manometer reading, H .

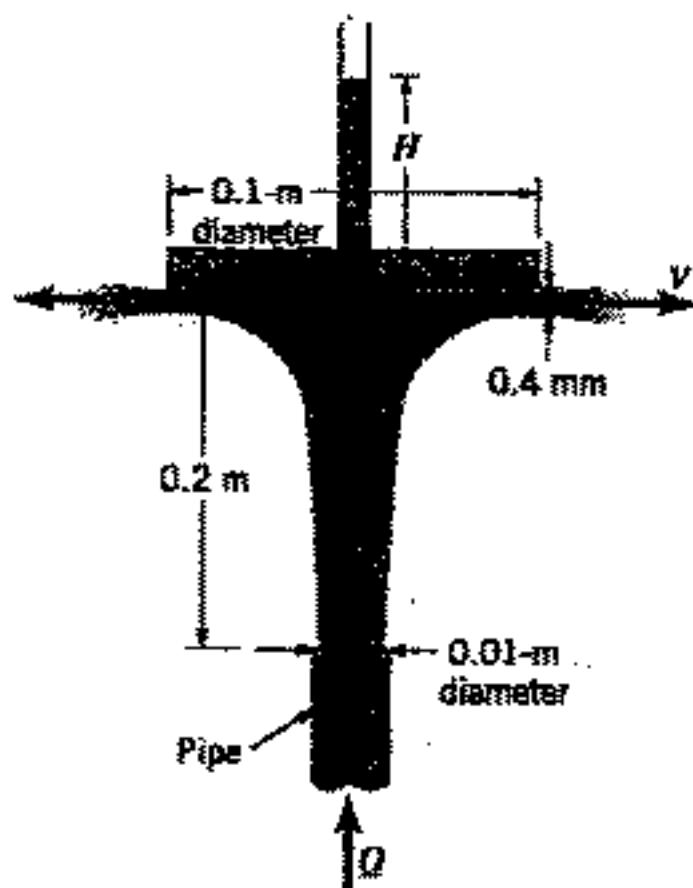
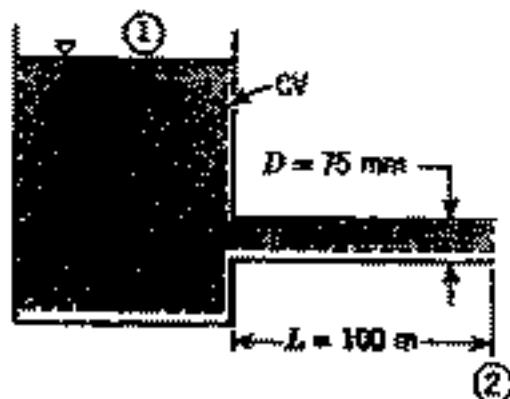


FIGURE III.

Prob. 4
25 points

A 100 m length of smooth horizontal pipe is attached to a large reservoir. What depth, d , must be maintained in the reservoir to produce a volume flow rate of $0.0084 \text{ m}^3/\text{s}$ of water? The inside diameter of the smooth pipe is 75 mm. The inlet is square-edged and water discharges to the atmosphere.



Hint:

Comparing equation:

$$\left(\frac{p_1}{\rho} + \alpha_1 \frac{\dot{V}_1^2}{2} + g z_1 \right) - \left(\frac{p_2}{\rho} + \alpha_2 \frac{\dot{V}_2^2}{2} + g z_2 \right) = h_{T2} \\ = h_T + h_{loss},$$

where

$$h_T = f \frac{L}{D} \frac{\dot{V}^2}{2} \quad \text{and} \quad h_{loss} = K \frac{\dot{V}^2}{2}.$$

$\cdot \rho = 1.0 \times 10^3 \text{ kg/m}^3$ for liquid water.

FIGURE IV.

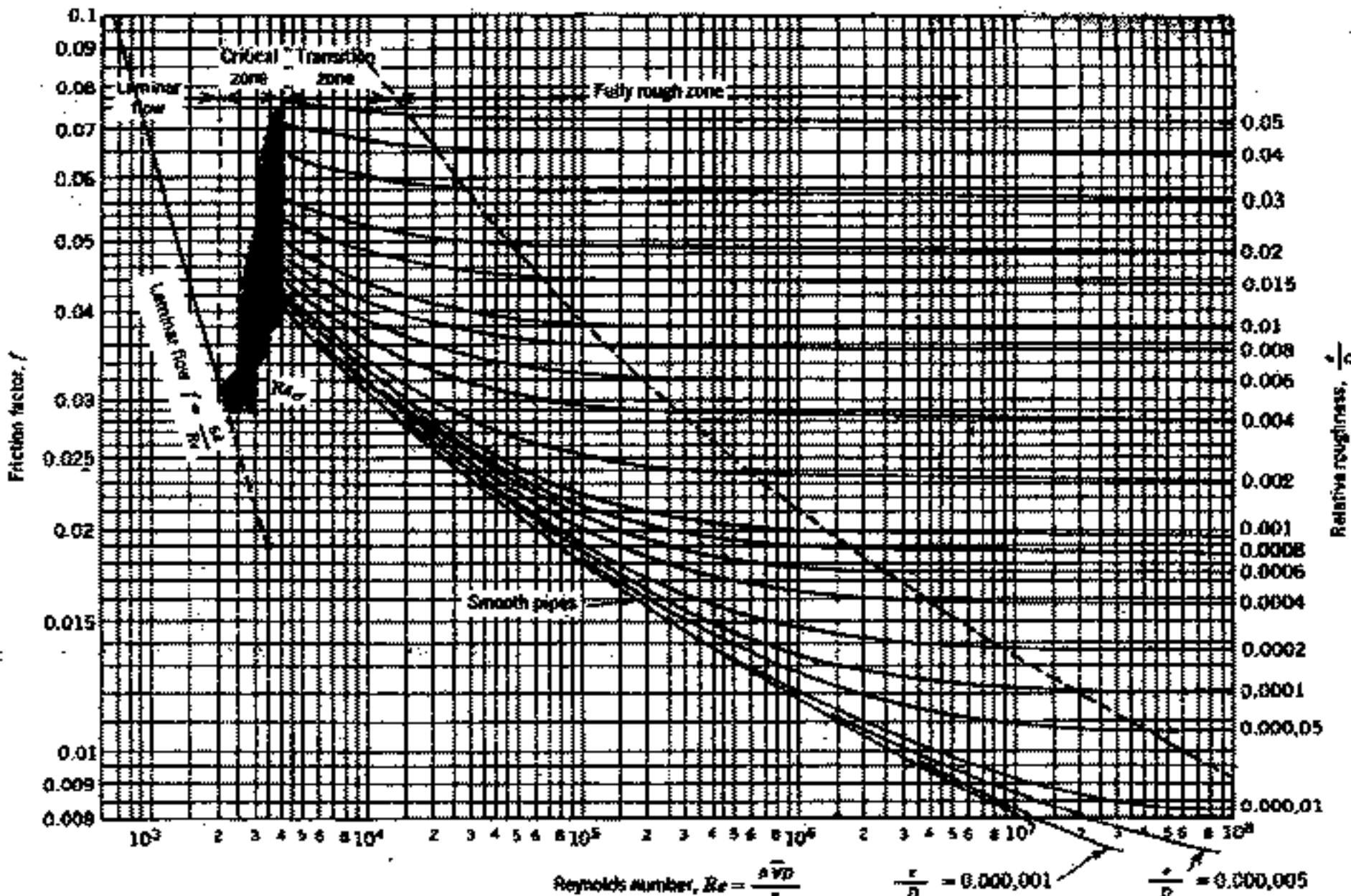


Fig. A1. Friction factor for fully developed flow in circular pipes. (Data from [6], used by permission.)

Table A1. Minor Loss Coefficients for Pipe Entrances (Data from [10].)

| Entrance Type | | Minor Loss Coefficient, K^* |
|---------------|--|--|
| Reentrant | | 0.78 |
| Square-edged | | 0.5 |
| Rounded | | $K = \frac{D}{r} \begin{cases} 0.02 & 0.06 \\ 0.28 & 0.15 \\ 0.04 & \geq 0.15 \end{cases}$ |

* Based on $h_L = K(\bar{V}^2/2)$, where \bar{V} is the mean velocity in the pipe.

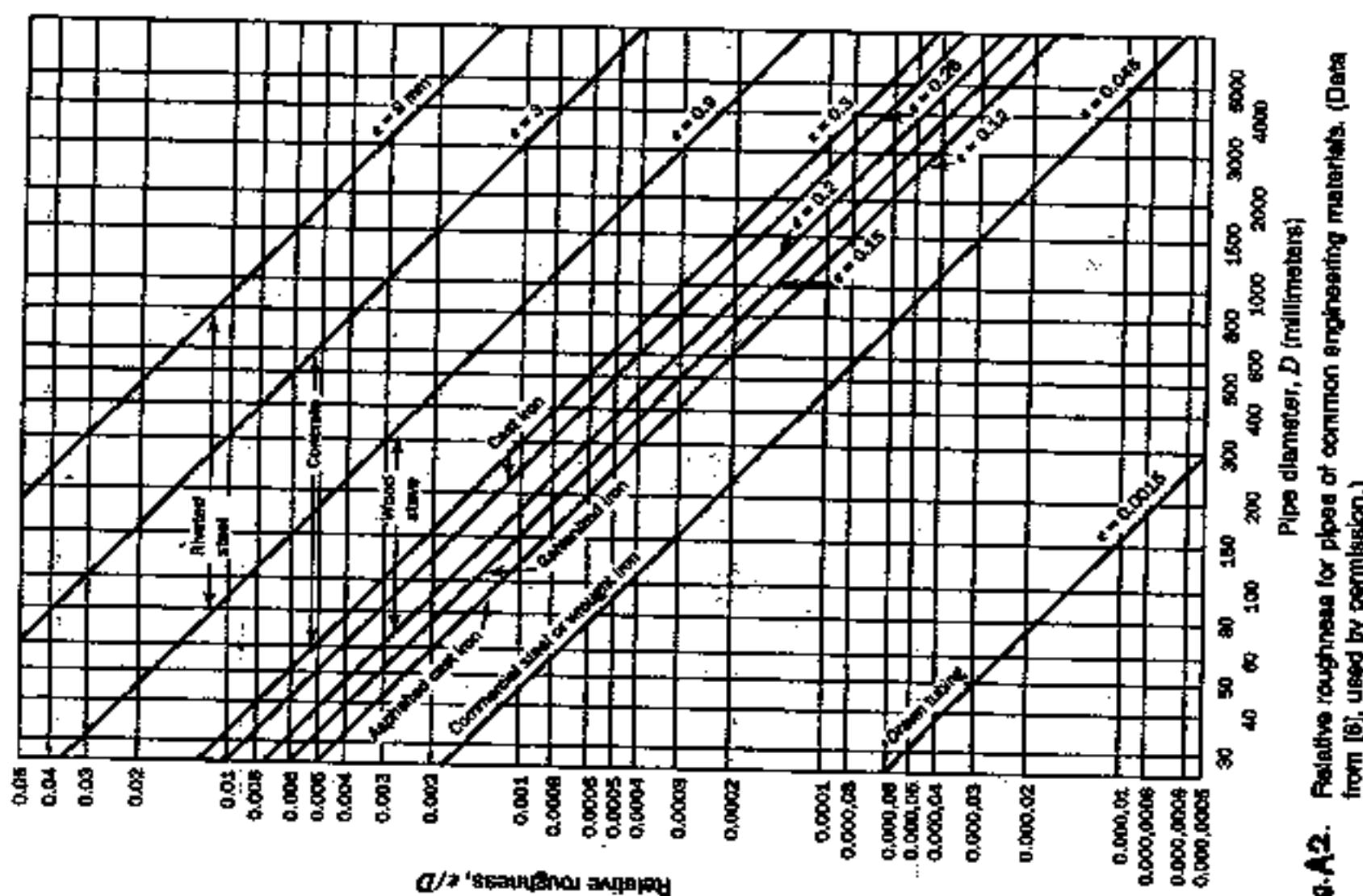


Fig. A.2. Relative roughness for pipes of common engineering materials. (Data from [8], used by permission.)

Table A.2. Representative Dimensionless Equivalent Lengths (L_e/D) for Valves and Fittings (Data from [10].)

| Fitting Type | Equivalent Length, L_e/D |
|---------------------------------------|----------------------------|
| Valves (fully open) | |
| Gate valve | 8 |
| Globe valve | 340 |
| Angle valve | 150 |
| Ball valve | 3 |
| Lift check valve: globe lift | 600 |
| : angle lift | 55 |
| Foot valve with strainer: poppet disk | 420 |
| : hinged disk | 75 |
| Standard elbow: 90° | 30 |
| : 45° | 16 |
| Return bend, close pattern | 50 |
| Standard tee: flow through run | 20 |
| : flow through branch | 60 |

* Based on $h_{L_e} = f \frac{L_e}{D} \frac{\bar{V}^2}{2}$.